Modeling of Radiative Heat Transfer of a Real Gas in 2D Enclosure

Mohammed Cherifi, Siham Laouar-Meftah and Abderahmane Benbrik
Laboratory of Hydrocarbon Physical Engineering
M'Hamed Bougara University
35000 Boumerdes, Algeria

Abstract

A two-dimensional radiation code based on the discrete ordinates method (DOM) coupled with spectral line-based weighted sum of grey gases (SLW) model for radiative heat transfer in non-grey absorbing–emitting media for use in conjunction with a computational fluid dynamics (CFD) code was developed. The code was applied to four different kinds of participating media: two homogeneous and isothermal and two non-homogeneous and non-isothermal cases with a single participating gas (CO2 or H2O). Predictive accuracy of the code is compared with the available results in the literature. Comparisons reveal that the discrete ordinates method (DOM) coupled with SLW model provides accurate solutions for radiative heat fluxes and source terms and can be used with confidence in conjunction with CFD codes.

Keywords
Natural convection, Double diffusion, Gas radiation, Discrete ordinates method, SLW model, 2D cavity.

1. Introduction

Radiative heat transfer plays an important role in many engineering problems, including atmospheric processes and combustion applications, namely, in furnaces, boilers, engines, rocket nozzles and fires (Kihwan Kim et al. 2008). The modeling of thermal radiation in these systems necessitates reliable evaluation of the medium radiative properties and accurate solution of the radiative transfer equation (RTE) in conjunction with the time-dependent conservation equations for mass, momentum, energy and chemical species. The computational effort associated with coupled solution of these governing equations can be minimized by using efficient and compatible solution techniques together with computationally feasible gas spectral radiative property models. In this paper, the radiative transfer equation (RTE) is solved by the discrete ordinates method (DOM) which is an attractive method as it provides efficient and flexible computation by enabling use of various higher-order approximations for temporal and spatial discretization. This method is associated with the spectral line weighted-sum-of-gray-gases (SLW) model which is an efficient model for prediction of radiative properties in high temperature gaseous media; the SLW model approaches the exact solution with an increase of the number of gray gases in the model (Vincent Goutière et al. 2000).

2. Modelling of radiation and gas radiative property

2.1 The radiative transfer equation

The radiative transfer equation RTE in an emitting-absorbing non-scattering medium may be written as

$$\frac{dI_{v,m}(x,y)}{dx} + \eta_m \frac{dI_{v,m}(x,y)}{dy} = -k_{\lambda}I_{v,m} + k_{\lambda}I_{b,m} \quad m = 1,\ldots,M$$

Where $I_{v,m}$ is the spectral radiation intensity, $k_{\lambda}$ is the spectral absorption coefficient of the medium, $s$ is the direction of propagation of radiation, and the subscripts $v$ and $b$ denote wave number and blackbody, respectively. The boundary condition for a gray surface that emits and reflects diffusely is given by:

$$I_{w,v}(\vec{s}) = \varepsilon_w I_{b,v} + \frac{1 - \varepsilon_w}{\pi} \int_{\vec{n} \cdot \vec{s} \leq 0} |\vec{n} \cdot \vec{s}'|I_{w,v}(\vec{s}')d\Omega'$$

Where $\varepsilon_w$ is the emissivity of the surface, $\vec{n}$ is the unit normal vector, $\vec{s}$ is the direction of the outgoing radiation intensity, and $\vec{s}'$ is the incoming direction associated with the elementary solid angle $d\Omega'$. 

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2.2 Gas radiative property model (SLW)

The motivation for using the SLW model is due to the work carried out by Goutière et al. (2000), who had compared several non gray methods in simple cases, where radiation is the dominant mode of heat transfer. Homogeneous as well as inhomogeneous media are considered. They judge the SLW method as the best, taking into account the computational cost, in terms of calculation time and memory requirements, and the quality of the results. However, the best results are obtained with the SNB and SNBCK models. The same conclusion, referring to the SLW, is stated by Coelho (2002). Furthermore, Bedir et al. (1997) compare both the SNB and SLW models for a one dimensional diffusion flame, and stated that the two models are in good agreement.

Denison and Webb (1993) introduced the Spectral Line Weighted sum of gray gases model, SLW, which can be considered as a refinement to the WSGG model, since it can be applied to inhomogeneous media.

In the SLW model, the non grey gas is replaced by a number of grey gases, each of which is represented with a constant absorption cross-section, \( \tilde{C}_{abs,j} \) and associated weight, \( a_j \). In this study, 15 absorption cross-sections logarithmically spaced between 3\( \times \)10\(^{-5} \) and 60 m\(^2\)/mol for water vapor and 3\( \times \)10\(^{-5} \) and 600 m\(^2\)/mol for carbon dioxide were utilized. These logarithmically spaced absorption cross-sections are called supplemental absorption cross-sections, \( \tilde{C}_{abs,j} \) as they are used to determine the grey gas weights, but do not appear directly in the corresponding grey gas absorption coefficient calculation. Each of the spaces between two consecutive supplemental absorption cross-sections, \( \tilde{C}_{abs,j} \) and \( \tilde{C}_{abs,j+1} \), is considered as a separate grey gas associated with a constant absorption cross-section calculated as follows:

\[
\tilde{C}_{abs,j} = \exp \left[ \frac{\ln \left( \tilde{C}_{abs,j} \right) + \ln \left( \tilde{C}_{abs,j+1} \right)}{2} \right]
\] (3)

Once the grey gas absorption cross-sections are obtained, local absorption coefficients are calculated from:

\[
k_{j} = N \cdot \tilde{C}_{abs,j}
\] (4)

The grey gas weights, \( a_j \), are calculated through absorption-line black body distribution functions derived from high-resolution HITRAN database. Correlations based on this function exist in the literature for the computation of the function \( F \) for CO\(_2\) (Denison and Webb, 1995), (Modest and Mehta, 2004), (Hongmei and Modest, 2003) and for H\(_2\)O (Denison and Webb, 1993).

3. Results and Discussion

To show the validity and the accuracy of the current code, different test problems are considered and compared with the available results of the literature.

3.1 Validation of the RTE solver

Before studying the real-gas cases, a test is made with a grey gas to ensure that the RTE solvers will not influence the results obtained with the real-gas models. The value of the grey gas absorption coefficient is taken as 0.5 m\(^{-1}\). The temperature of the medium is kept uniform at 1000 K and the walls are black and cold. Figure 1 shows that the results yielded by the three RTE solvers are in good agreement.
Figure 1: (a) Evolution of the heat flux $q_x$ for the grey case; (b) Evolution of the heat flux $q_y$ for the grey case; (c) Evolution of the source term $-(\text{div} q)_x$ for the grey case; (d) Evolution of the source term $-(\text{div} q)_y$ for the grey case.

As can be observed in Fig.2, the results yielded by the RTE solver is in good agreement.

3.2 Validation of the RTE solver associated with SLW

In this study, we follow the same steps as in the reference Vincent Goutière et al. (2000). The enclosure used for all the tests is rectangular (1x0.5 m) and its walls are black and kept at 0 K (Figure 2). The grid is uniform (61x31) and the $S_{16}$ quadrature with 178 directions is used ($T_7$ quadrature with 392 directions was used in the reference of Vincent Goutière et al. (2000). In order to make meaningful comparisons, we have chosen four different kinds of participating media: two homogeneous and isothermal cases with a single participating gas (CO2 and H2O), two non-homogeneous and non-isothermal cases with a single participating gas (CO2 and H2O). For all the four cases, the non-participating gas is N2 and the pressure of the gas medium is 1 atm.
3.2.1 Homogeneous and isothermal medium
The cavity is filled with either CO₂ (10%) (Case 1) or H₂O (20%) (Case 2), and maintained at 1000K. The wall heat flux and the divergence of heat flux are calculated.

![Figure 2: The enclosure](image1)

![Figure 3: (a) Evolution of the heat flux \( q_x \); (b) Evolution of the heat flux \( q_y \); (c) Evolution of the heat flux \(-\delta q_x/\delta x\); (d) Evolution of the heat flux \(-\delta q_y/\delta y\);](image2)
Figure 4: (a) Evolution of the heat flux $q_x$.
(b) Evolution of the heat flux $q_y$.
(c) Evolution of the source term $-(\text{div} q)_x$.
(d) Evolution of the source term $-(\text{div} q)_y$.

3.2.2 Non homogeneous and non isothermal medium

The cavity is the same as in the cases above, this time with variable temperature and concentration of the radiating gases, CO$_2$ or H$_2$O, according to the following functions. The temperature and the concentration are:

$$T(x, y) = T_0 \left[ 0.3333 \left( 1 - 2 \left| x - 0.5 \right| \right) \left( 1 - 4 \left| y - 0.25 \right| \right) + 1 \right]$$

(5)

$$c(x, y) = c_0 \left[ 4 \left( 1 - 2 \left| x - 0.5 \right| \right) \left( 1 - 4 \left| y - 0.25 \right| \right) + 1 \right]$$

(6)

Where $T_0$ is 1200 K and $c_0$ is 0.02 for CO$_2$ (case 3) and 0.04 for H$_2$O (case 4).

The mesh used is the same of that one used in the previous cases. The reference temperature from which the effective absorption coefficient is implicitly determined has some impact on the final result. For this particular case, the reference temperature has been taken to be the volumetric mean and a similar criterion has been used to define the reference concentration for the radiating gas case.
Figure 5: (a) Evolution of the heat flux $q_x$
(b) Evolution of the heat flux $q_y$
(c) Evolution of the source term $-(\text{div } q)x$
(d) Evolution of the source term $-(\text{div } q)y$
for the case 3 (non Homogeneous, non isothermal medium, CO$_2$).
As it can be observed from figures 3-5, a good agreement between the calculated values and those reported in the reference of Vincent Goutière et al. (2000) with a small difference for the heat flux divergence. The greatest difference is in the calculated wall heat flux which is probably due to the different reference concentration and temperature used in both calculations and to the ray effects due to the quadrature type (S16 quadrature with 178 directions is used in this study and T7 quadrature with 392 directions was used in the reference of Vincent Goutière et al. (2000)).

5. Conclusions
A two-dimensional radiation code based on the discrete ordinates method (DOM) coupled with spectral line-based weighted sum of grey gases (SLW) model for radiative heat transfer in non-grey absorbing–emitting media for use in conjunction with a computational fluid dynamics (CFD) code was developed. Comparisons reveal that the discrete ordinates method (DOM) coupled with SLW model provides accurate solutions for radiative heat fluxes and source terms and can be used with confidence in conjunction with CFD codes.

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References


Biography
Mohammed Cherifi is Ph.D. student. He earned his Master degree in 2009. The subject of his thesis is devoted to the Combined Effect of Radiation and Natural Convection in a Square Cavity.

Siham Laouar-Meftah holds an engineering degree in energy from the university Larbi Ben Mhidi of Oum El Bouaghi (Algeria, 1993). She earned a Magister degree in energy from National Polytechnic School of Algiers (Algeria, 1998) and a Ph.D. degree in Mechanical Engineering (option: thermo-fluids) from the University of M’Hamed Bougara of Boumerdes (Algeria, 2010). Now, she teaches permanently at Faculty of Hydrocarbons and Chemistry of University of Boumerdes and is a member of a team of researchers in the research laboratory of hydrocarbon physical engineering.

Abderrahmane Benbrik is currently Professor at the Department of Transport and Hydrocarbon Equipments, Faculty of Hydrocarbons and Chemistry at M’Hamed Bougara University in Boumerdes (Algeria). He earned a Bachelor degree in Mechanical Engineering (Major : Drilling Machinery) from the Institute of Hydrocarbons and Chemistry of Boumerdes (Algeria) and a Ph.D. degree in Technical Sciences from the GOUBKINE Institute of Petrol and Gas, Moscow, Russia. He is team leader of research in the field of modeling and optimization of thermal systems in the research laboratory of hydrocarbon physical Engineering. He has taught courses of heat transfer, thermodynamic, thermal engines and drilling equipments for engineers. He has been an external examiner of several Master and PhD thesis in the area of thermo-fluids engineering.